



Dominator 140 Dominator 150

Technical Systems

Hydraulic System



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Chapter 1

Overall hydraulic system

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1.1

Overall hydraulic system circuit diagram

with 3D sieve pan

1.1 Overall hydraulic system circuit diagram with 3D sieve pan



I II IV V VI	Master valve working hydraulics valve block Front attachment / threshing drum variator working hydraulics valve block Grain tank unloading tube working hydraulics valve block Front attachment reverse working hydraulics valve block Orbitrol steering hydraulics Ground drive hydraulics hydrostatic pump Ground drive hydraulics hydrostatic pump
102 107 109 110 112	Pressure filter
205 207 209 211 218	Working hydraulics pump 10.8 cm³/rev. Ground drive fixed displacement motor HMF 75 75 cm³/rev. Ground drive feed pump 22.5 cm³/rev. Ground drive variable displacement pump HPV 75 75 cm³/rev. Steering hydraulics pump 6 cm³/rev.
220 301 311 313 314 315 316 320 323 351 353	3-D sieve pan hydraulic cylinder Threshing drum variable-speed drive hydraulic cylinder Ground drive servo control pump hydraulic cylinder Reel raise/lower slave cylinder Horizontal reel adjustment hydraulic cylinder Swing grain tank unloading tube hydraulic cylinder Steering hydraulic cylinder Raise/lower front attachment hydraulic cylinder Reverse front attachment hydraulic cylinder
406 410 441	Orifice plateØ 0.8 mm Orifice plateØ 1.5 mm Rotary coupling
601 606 609 614	3D sieve pan pendulum control 4/3 way valve Ground drive servo control 4/3 way valve Orbitrol steering system rotary valve Front attachment lower flow control valve

Key to diagram:

703	Working hydraulics pressure relief valve
710	Ground drive filter bypass valve 2 bar
713	Ground drive multi-function valve, reverse
714	Ground drive multi-function valve, forward
716	Ground drive feed pressure relief valve
718	Ground drive feed circuit cold start injector
719	Ground drive flush pressure control valve
721	Ground drive flush-out shuttle valve
725	Steering double shock valve 150 ⁺¹⁵ bar
726	Steering pressure relief valve
728	Anti-cavitation valve (non-return valve)
732	Non-return valve
734	Lock-up valve unit (non-return valve)
742	Steering safety valve
743	Front attachment lower valve
759	One-way restrictor valve, two-sided
801	Quick release coupling
901	Working hydraulics measuring point
902	Ground drive hydraulics high pressure backward measuring point
903	Ground drive hydraulics high pressure forward measuring
	point
904	Ground drive hydraulics feed pressure measuring point
909	Steering hydraulics measuring point
Y19	Threshing drum variable-speed drive slow solenoid valve
Y20	I hreshing drum variable-speed drive fast solenoid valve
Y22	Reel raise solenoid valve
Y23	Reel lower solenoid valve
Y24	Reel forward solenoid valve
120 V22	Reel reverse solenoid valve
100 V3/	Grain tank unloading tube swing out solehold valve
V77	Working hydraulice master valve solenoid valve
Y85	Raise front attachment solenoid valve
Y86	Reverse front attachment solenoid valve
Y87	Lower front attachment solenoid valve

1.2

Overall hydraulic system circuit diagram

without 3D sieve pan

1.2 Overall hydraulic system circuit diagram without 3D sieve pan



I II IV V VI VI	Master valve working hydraulics valve block Front attachment / threshing drum variator working hydraulics valve block Grain tank unloading tube working hydraulics valve block Front attachment reverse working hydraulics valve block Orbitrol steering hydraulics Ground drive hydraulics hydrostatic pump Ground drive hydraulics hydrostatic motor
102 107 109 110 112	Pressure filter
205 207 209 211 218 226	Working hydraulics pump10.8 cm³/rev.Ground drive fixed displacement motor HMF 75 75 cm³/rev.Ground drive feed pump22.5 cm³/rev.Ground drive variable displacement pumpHPV 7575 cm³/rev.Steering hydraulics pump6 cm³/rev.Front attachment reverser drive motor
311	Threshing drum variable-speed drive hydraulic cylinder
313	Ground drive servo control pump hydraulic cylinder
314	Reel raise/lower slave cylinder
315	Reel raise/lower main cylinder
316	Horizontal reel adjustment hydraulic cylinder
320	Swing grain tank unloading tube hydraulic cylinder
323	Steering hydraulic cylinder
351	Raise/lower front attachment hydraulic cylinder
353	Reverse front attachment hydraulic cylinder
406	Orifice plateØ 0.8 mm
410	Orifice plateØ 1.5 mm
441	Rotary coupling
606	Ground drive servo control 4/3 way valve
609	Orbitrol steering system rotary valve
614	Front attachment lower flow control valve

04/04

Key to diagram:

703	Working hydraulics pressure relief valve
710	Ground drive filter bypass valve
713	Ground drive multi-function valve, reverse
714	Ground drive multi-function valve, forward
716	Ground drive feed pressure relief valve
718	Ground drive feed circuit cold start injector
719	Ground drive flush pressure control valve
721	Ground drive flush-out shuttle valve
725	Steering double shock valve150 ⁺¹⁵ bar
726	Steering pressure relief valve90 ⁺⁵ bar
728	Anti-cavitation valve (non-return valve)
732	Non-return valve
734	Lock-up valve unit (non-return valve)
742	Steering safety valve
743	Front attachment lower valve
759	One-way restrictor valve, two-sided
801	Quick release coupling
901	Working hydraulics measuring point
902	Ground drive hydraulics high pressure backward measuring
903	Ground drive hydraulics high pressure forward measuring
	point
904	Ground drive hydraulics feed pressure measuring point
909	Steering hydraulics measuring point
Y19	Threshing drum variable-speed drive slow solenoid valve
Y20	Threshing drum variable-speed drive fast solenoid valve
Y22	Reel raise solenoid valve
Y23	Reel lower solenoid valve
Y24	Reel forward solenoid valve
Y25	Reel reverse solenoid valve
Y33	Grain tank unloading tube swing out solenoid valve
Y34	Grain tank unloading tube swing in solenoid valve
Y77	Working hydraulics master valve solenoid valve
Y85	Raise front attachment solenoid valve
Y86	Reverse front attachment solenoid valve
Y87	Lower front attachment solenoid valve

1.3 Technical data			
Working hydraulics	Dominator 140 -150	Sense of rotation Drive n _{max} Power	= Counterclockwise = 3092 rpm = 33 l/min at 10.8 cm ³ /rev.
Steering hydraulics	Dominator 140 -150	Sense of rotation Drive n _{max} Power	= Counterclockwise = 3092 rpm = 18.5 l/min at 6 cm ³ /rev.
Linde ground drive	Dominator 140 -150	Sense of rotation Drive n _{max} Power approx.	= Counterclockwise = 3092 rpm = 230 l/min at 75 cm ³ /rev.
Linde feed pump	Dominator 140 -150	Sense of rotation Drive n _{max} Power	= Counterclockwise = 3092 rpm = 69.5 l/min at 22.5 cm ³ /rev.
	Note: The exact pump of During this measu pressure and at of 15% max.	output can only be c urement, the output perating temperatu	letermined using a flowmeter. under maximum system re (60°C) may fall by

Chapter 2

Steering hydraulics

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2.1

Steering hydraulics circuit diagram

2.1 Steering hydraulics circuit diagram



Key to diagram:	V VI	Orbitrol steering hydraulics Ground drive hydraulics hydrostatic pump
	107 110	Oil drain Oil tank
	205 209 211 218	Working hydraulics pump Ground drive feed pump Ground drive variable displacement pump Steering hydraulics pump
	323	Steering hydraulic cylinder
	609	Orbitrol steering system rotary valve
	718 725 726 728 742	Ground drive feed circuit cold start injector
	909	Steering hydraulics measuring point
Pressure measurement :	Neutra Syste Shock	al circulation pressure = < 20 bar m pressure = 90 ⁺⁵ bar c valve = 150 ⁺¹⁵ bar
	Note:	These values refer to measurements made at the max. no-loa

ad speed of the diesel engine and a hydraulic oil operating temperature of approx. 60°C.

as the steering wheel.

Description of function:

Steering

In the neutral steering position, oil flows freely through the steering control unit = Orbitrol (609). Turning the steering wheel to one direction causes the spools to rotate relative to each other. At a rotation of 1.5° , the channels to the chambers start opening. At 4°, the neutral position channels are completely closed. At 6°, the channels to the chambers are fully open. The rotation of the spools relative to each other is limited to $\pm 8^{\circ}$. A feed of pressurized oil to the rotor set has the following effects: Rotation of rotor: Feed of an oil quantity which is proportional to the rotation into the steering cylinder, the rear wheels being influenced. An internal mechanical return from the rotor to the outside spool so that the

channels in the valve are closed when the rotor rotates to the same angle

2.2 Steering valve unit



Key to diagram:	112Return218Steerir229Steerir323Steerir609Orbitro725Steerir726Steerir728Anti-ca742Steerir909Steerir	filter (not installed) ng hydraulics pump	
Steering system	Open center:	with the steering in neutral position, there is a connection between pump P2 and the tank.	
	Non reaction:	when the steering is in neutral position, a force acting on the steered wheels does not cause any reaction on the steering wheel.	
Valve unit	DANFOSS OSI	PB 125	
	O = Orbit (Orbit S = Steering P = Pump B = Version 125 = Oil displa	rol) icement in cm³/rev.	
Design of valve unit	The steering valve consists of a steering hydraulics proportioning pump (229) and an Orbitrol rotary valve (609).		
	The Orbitrol rota Continued rotar hydraulics prop	ary valve (609) is actuated by the steering gear shaft. y movement of the steering gear shaft drive the steering ortioning pump (229) by means of a socket-type shaft.	

2.3 Function of steering Neutral



DO-h-Kap2

Description of function:

Neutral

In neutral position, the oil is directed back to the tank via the steering safety valve (742) and the Orbitrol rotary disc (609) (open center). The circulation pressure must not exceed 20 bar. Both sides of the steering hydraulic cylinder (323) are shut off by the Orbitrol rotary disc (609). Pressure peaks due to external forces on the steered axle are relieved to the tank via the steering double shock valves (725) (non reaction).

Steering actuation



Description of function:

Steering actuation	When actuating the steering to one or another direction, the Orbitrol rotary disc (609) is rotated by up to 8° relative to the outside spool. During this process, the return line from the steering hydraulics pump (218) to the tank is closed and the connection to the steering hydraulics proportioning pump (229) is released.			
	Via the steering hydraulics proportioning pump (229) and the Orbitrol rotary disc (609), the volume flow is released as a function of the sense of rotation, path and speed of steering wheel motion to the ram or the ram ring surface of the steering hydraulic cylinder (323). Here, the displacing surface of the steering hydraulic cylinder (323) is connected with the return line to the tank via the Orbitrol rotary disc (609).			
	As soon as there is no more steering motion, leaf springs bring the outer rotary disc of the Orbitrol rotary disc (609) back to neutral position. Now both sides of the steering cylinder are shut off again and the connection from the steering hydraulics pump (218) to the tank is re-established.			
Emergency steering	When the steering system is not supplied any more by the steering hydraulics pump (218), the steering safety valve (742) closes and thus ensures that no oil will escape from the steering system.			
	When the steering is actuated, the inner and outer disc of the Orbitrol rotary disc (609) are rotated relative to each other. Now the oil can be conveyed from one side of the steering hydraulic cylinder (323) via anti-cavitation valve (non-return valve) (728) to the other side through human power by the drive of the steering hydraulics proportioning pump (229).			

2.4 Checking the steering			
Steering gear shaft	Height play = 0.1 to 0.3 mm		
	Clearance fr	om bottom inside rotary disc = 3 mm	
Return	When the ste leaf springs neutral posit	eering wheel is actuated with the diesel engine shut off, the in the rotary disc must bring the steering wheel back to its ion.	
Reaction	If steering reaction is insufficient, internal leaks in the steering system must be checked. To do this, disconnect the lines from the steering cylinder and plug them tightly with plugs. With the oil at operating temperature and at max. no-load speed of the diesel engine, the steering wheel must not allow more than 4 turns/minute in both directions when using a force of approx. 25 Nm. When the actual number of turns is more than 4/minute, check the steering valve for leaks. When the actual number of turns is below 4/minute, check the steering cylinder for leaks.		
Power In case of steering forces above 25 Nm, cl condition of the cylinder rod and if stub axl A pressure test at the steering hydraulics r value 90 ⁺⁵ bar. To this end move the steering wheel up to position. Adjusting the pressure relief valve on the r not possible.		eering forces above 25 Nm, check tyre size and pressure, the cylinder rod and if stub axles move smoothly. est at the steering hydraulics measuring port must show the ar. move the steering wheel up to the stop and hold it in this e pressure relief valve on the machine in built-in condition is	
	Important!	Any installation work on the steering hydraulics must be followed by venting the system on both hydraulic lines of the steering cylinder with the diesel engine running.	

Chapter 3

Working hydraulics

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	3/2-way valve Y86	
	Hydraulic cylinders	

3.1

Working hydraulics circuit diagram

- without straw collector

3.1 Working hydraulics circuit diagram - without straw collector



I II IV VI	Master valve Front attachr valve block Grain tank ur Front attachr Ground drive
107 110 112	Oil drain Oil tank Return filter
205 209 211	Working hydr Ground drive Ground drive
218 226	Steering hyd Front attachr
311 314 315 316 320 351 353	Threshing dr Reel raise/lov Reel raise/lov Horizontal re Swing grain t Raise/lower t Reverse from
406 441	Orifice plate. Rotary coupl
614	Front attachr
703 718 732 734 743 759	Working hyd Ground drive Non-return va Lock-up valv Front attachr One-way res
801	Quick release
901	Working hyd
Y19 Y20 Y22 Y23 Y24 Y25 Y33 Y34 Y77 Y85 Y86 Y87	Threshing dr Threshing dr Reel raise so Reel lower so Reel forward Reel reverse Grain tank ur Grain tank ur Working hyd Raise front a Reverse fron Lower front a

TIC

Key to diagram:

e working hydraulics valve block ment / threshing drum variator working hydraulics

nloading tube working hydraulics valve block ment reverse working hydraulics valve block e hydraulics hydrostatic pump

rum variable-speed drive hydraulic cylinder ower slave cylinder ower master cylinder eel adjustment hydraulic cylinder tank unloading tube hydraulic cylinder

front attachment hydraulic cylinder nt attachment hydraulic cylinder

.....Ø 0.8 mm ling

ment lower flow control valve

se coupling

raulics measuring point

rum variable-speed drive slow solenoid valve rum variable-speed drive fast solenoid valve olenoid valve solenoid valve d solenoid valve e solenoid valve unloading tube swing out solenoid valve unloading tube swing in solenoid valve draulics master valve solenoid valve attachment solenoid valve

attachment solenoid valve

Description of function:

The working hydraulics of the Dominator series is an open hydraulic system.

The maximum system pressure is limited to 180^{+10} bar by means of pressure relief valve (703).

3.2 Main valve

with master valve, pressure relief valve, raise/lower front attachment



Key to diagram:	614 703 732 743	Flow control valve
	Y77 Y85 Y87	Master valve solenoid valve Raise front attachment solenoid valve Lower front attachment solenoid valve
	A1 P1 P2 T	Raise/lower front attachment hydraulic cylinder Working hydraulics pump port Parallel port for working hydraulics of other directional control valves Tank port
	E F H K L R U V	Pilot spool Compression spring Front attachment raise spool Front attachment quick lower ram Lower front attachment spool Control spool

Description of function:

Pressure limitation

The spring in the pressure relief valve (703) is pre-stressed for a system pressure of 180^{+10} bar. The pressure setting may be modified by removing or adding shims.

- A 0.5 mm shim corresponds to approx. 10 bar
- A 1.2 mm shim corresponds to approx. 23 bar
- **Note:** The above values refer to a rated pressure of 180 bar and may deviate, depending on the actual system. Each time the setting has been modified, the system pressure must be checked.

Basic setting



To ensure the position of spool (H) for the function "Raise front attachment", the dimension from the top edge of the spool (H) to the body must be **4.3 mm** with the coil core (Y85) removed. The position may be corrected by removing or adding shims above the compression spring.

A weaker compressed spring - as compared to the spool (H) - is located below the spool of the master valve (U).



In order to guarantee the "Front attachment lower" function, the clearance between the top edge of spool (L) and the housing must be **4.5 mm** with the coil core (Y87) removed. The position may be corrected by removing or adding shims.

Spare part no: 0.1 mm = 0218 886.0 0.2 mm = 0218 887.0

The drop rate of the front attachment can be adjusted to a drop time of **5-6 sec.** over the entire stroke range.on the flow control valve (614).
TIC	Γ	Dominator 140 - 150	Hydraulic System
Key to diagram:	P M R(T)	Pump port Measuring port Return line port (tank)	
Description of function:			
Pressure relief valve	The pres connecte The spri pressure or addin	ssure relief valve protects the hydr ed mechanical components from d ng in the pressure relief valve (703 of 180 ⁺¹⁰ bar. The pressure settir g shims.	aulic system and thus the lamage by excessive forces. 3) is pre-stressed for a system ng may be modified by removing

A 0.5 mm shim corresponds to approx. 10 bar A 1.2 mm shim corresponds to approx. 23 bar

temperature of approx. 60°C.

Note: The above values refer to a rated pressure of 180 bar and may deviate, depending on the actual system. Each time the setting has been modified, the system pressure must be checked.

Note: These values refer to measurements made at the max. no-load speed of the diesel engine and a hydraulic oil operating

D

Description of function:	
Function of master valve	The master valve (Y77) blocks the circulating volume flow from P to T of the open hydraulic system when a working hydraulics function has been actuated. Single-acting functions are an exception to this if the consumer is relieved to the tank.
	In neutral position, the master valve (Y77) is not actuated, making the oil flow back to the tank via the ring channels on the spool (U). Due to the large channel cross-section, the circulation pressure is very low.
	When pressure is successfully built up at a consumer, the master valve (Y77) is actuated simultaneously with the directional control valve of the corresponding function. Now spool (U) closes the connection from P to T, and the top ring channel being closed first in order to achieve smooth switching-over.
	The pressure relief valve (703) opens at a maximum system pressure of 180±10 bar and relieves the pressure to the tank.
Raise front attachment function	
Raising	When the "Raise" function is used, the directional control valve (Y85) and the master valve (Y77) are actuated with 12 V DC.
	The spool (H) is moved to its end position, making oil flow via both ring channels on the spool (H). The full volume flow is directed to the consumer port (A1) via the non-return valve (732) and raises the front attachment.
Lower front attachment function	
Lowering	When the "Lower" function is used, only the directional control valve (Y87) is actuated with 12 V DC. The spherical seat in pilot valve (V) is opened and the spring force (F) is overcome through the force of the solenoid. The spool (E) closes the ring channel to the return line, making the load pressure of the front attachment act on the ram top side (K) and open the pilot valve (743). The oil is now displaced by the front attachment via the pilot valve (743) and the control spool (R) into the return line to the tank.

Description of function:

Flow control valve

When the "Lower front attachment – fast" function is used, the oil displaced via port A flows to the tank (T) through the restrictor in the control spool of the flow control valve (614).

This creates a ram pressure ahead of the control spool, making the latter move against the control spring and restrict the return channel to the tank (T) as a function of the load pressure.

When the load pressure in port A changes, both the volume flow through the restrictor and the load pressure against the control spool change, too, and consequently also the return channel cross-section.

This control function keeps the volume flow and therefore the front attachment drop rate constant, independent of the load pressure. The front attachment drop rate is adjusted merely by the pre-stress of the control spring at the handwheel.

Relieve tension of control spring = lower drop rate Tensioning the control spring = increase drop rate

Threshing drum speed control 3/3 way solenoid valve Y19/Y20 3.3



Key to diagram:	II	Front attachment / threshing drum variator working hydraulics valve block	
	311 314 315 316	Threshing drum variable-speed drive hydraulic cylinder Reel raise/lower slave cylinder Reel raise/lower master cylinder Horizontal rool adjustment hydraulic cylinder	
	310		
	441	Rotary coupling	
	759	One-way restrictor valve, two-sided	
	801	Quick release coupling	
	Y19 Y20 Y22 Y23 Y24 Y25	Threshing drum variable-speed drive slow solenoid valve Threshing drum variable-speed drive fast solenoid valve Reel raise solenoid valve Reel lower solenoid valve Reel forward solenoid valve Reel reverse solenoid valve	
Description of function:			
Neutral function	The threshing drum variable-speed drive hydraulic cylinder (311) is tightly closed by the ball seat in the valve insert of the threshing drum slow solenoid valve (Y19).		
Increase speed function	The thre actuated the valve rising op slow sole in the on	The threshing drum fast solenoid valve (Y20) and the master valve are actuated at the same time. The corresponding pilot spool opens the ball in the valve insert and closes the return line to the tank. The pressure thus rising opens the ball in the valve insert of the unactuated threshing drum slow solenoid valve (Y19). The oil flows to consumer port A1 via the notch in the one-way restrictor valve (759).	
Reduce speed function	Solenoic in questi pressure insert of tank.	d valve (Y19) is actuated without the master valve. The pilot spool on opens the ball in the valve insert and thus relieves the oil e via the notch of the one-way restrictor valve (759) and the valve the unactuated threshing drum fast solenoid valve (Y20) to the	
	Note:	To ensure even control function in both directions, volume flow flows via the notches in the one-way restrictor valve (759) when adjusting the variator.	

Threshing drum speed control Hydraulic cylinder with rotary coupling



TIC		Dominator 140 - 150	Hydraulic System
Key to diagram:	311	Threshing drum variable-spee	d drive hydraulic cylinder
	441	Rotary coupling	

Vertical reel adjustment 3/3 way valve Y22/Y23 3.4



Key to diagram:	II	Front attachment / threshing drum variator working hydraulics valve block		
	311 314 315 316	Threshing drum variable-speed drive hydraulic cylinder Reel raise/lower slave cylinder Reel raise/lower master cylinder Horizontal reel adjustment hydraulic cylinder		
	441	Rotary coupling		
	759	One-way restrictor valve, two-sided		
	801	Quick release coupling		
	Y19 Y20 Y22 Y23 Y24 Y25	Threshing drum variable-speed drive slow solenoid valve Threshing drum variable-speed drive fast solenoid valve Reel raise solenoid valve Reel lower solenoid valve Reel forward solenoid valve Reel reverse solenoid valve		
Description of function:				
Neutral	The hydr valve (Y2	The hydraulic cylinders are tightly closed by the valve insert of solenoid valve (Y23).		
Raise reel	The soler same tim insert and The pres unactuate port A2.	The solenoid valve (Y22) and the master valve (Y77) are actuated at the same time. The corresponding pilot spool opens the ball in the valve insert and closes the return line to the tank. The pressure P1 which consequently rises opens the valve insert of the unactuated solenoid valve (Y23) and the oil flows to the consumer port A2.		
Lower reel	Solenoid spool in c	Solenoid valve (Y23) is actuated without the master valve (Y77). The pilot spool in question opens the ball in the valve insert and thus relieves the		

valve (Y22).

oil pressure to the tank via the valve insert of the unactuated solenoid



Key to diagram:	314 315	Reel raise/lower slave cylinder Reel raise/lower master cylinder
	A3 E V	Hydraulic cylinder port Vent plug Bottom valves

Description of function:

Bottom valves

The bottom outlet valves (V) in master cylinder (315) are opened upon reaching the upper stop position so that the slave cylinder can be filled and vented.

Note: For repairs, it is recommended to remove the hydraulic rams in the raised reel position since the slave cylinder is filled only with the master cylinder fully extended. During this process support and secure the reel properly.

Horizontal reel adjustment 4/3 way valve Y24/Y25 3.5



Key to diagram:	II	Front attachment / threshing drum variator working hydraulics valve block
	311 314 315 316	Threshing drum variable-speed drive hydraulic cylinder Reel raise/lower slave cylinder Reel raise/lower master cylinder Horizontal reel adiustment hydraulic cylinder
	441	Rotary coupling
	759	One-way restrictor valve, two-sided
	801	Quick release coupling
	Y19 Y20 Y22 Y23 Y24 Y25	Threshing drum variable-speed drive slow solenoid valve Threshing drum variable-speed drive fast solenoid valve Reel raise solenoid valve Reel lower solenoid valve Reel forward solenoid valve Reel reverse solenoid valve

Description of function:

Neutral	Both sides of the hydraulic cylinder are tightly closed by the lock-up valve unit (734).
Reel forward / reverse	Depending on the necessary direction of movement, one of the solenoid valves (Y24/Y25) and, at the same time, the master valve (Y77) is actuated. The corresponding pilot spool opens the ball in the valve insert and closes the return line to the tank. The pressure which consequently rises builds up against the ram in lock-up valve unit (734) and thus unlocks the return line to the tank in the opposite port. The return line of the hydraulic cylinder is relieved to the tank via the valve insert of the unactuated solenoid valve (Y24/Y25). The pressure rising further now opens the lock-up valve unit (734) on the pressure side and the hydraulic cylinders are retracted or extended.

Horizontal reel adjustment Hydraulic cylinders



Key to diagram:	316	Horizontal reel adjustment hydraulic cylinder
	A2 A3 V	Hydraulic cylinder port Hydraulic cylinder port Bottom valves

Description of function:

Bottom valves

The bottom outlet valves (V) open every time an end position is reached so that air inclusions in the connection between the two rams can be flushed out.

After a repair, the cylinders must be flushed in both end positions for approx. 15 sec.

Horizontal reel adjustment Lock-up valve unit (734)



Key to diagram:

- A Hydraulic valve port
 B Hydraulic valve port
 A1 Consumer port
 B1 Consumer port
 - B1 Consumer port

Description of function:

Lock-up valve units (pilot controlled non-return valves) are used in order to lock functions while pressure is relieved and thus to ensure a fixed position of a consumer.

A rising pressure in port (B) moves the internal ram (K). This opens the opposite non-return valve in port A - the return line of the hydraulic cylinder to the tank is relieved (connection A-A1).

The continued pressure increase now opens the non-return valve in port B. The connection to consumer (B-B1) is relieved.

3.6 Swinging the grain tank unloading tube 4/3 way valve Y33/Y34



Key to diagram:	III	Grain tank unloading tube working hydraulics valve block		
	Y33 Y34	Grain tank unloading tube swing out solenoid valve Grain tank unloading tube swing in solenoid valve		
	A B P1 T	Consumer port Consumer port Working hydraulics pump port Tank port		
Description of function:				
Neutral function	Both side return va	Both sides of the hydraulic cylinder (320) are tightly closed by the non-return valves (734).		
Description of function	Dependii valves (Y actuated and close The pres up valve cylinder	ng on the necessary direction of movement, one of the solenoid (33/Y34) and, at the same time, the master valve (Y77) is . The corresponding pilot spool opens the ball in the valve insert es the return line to the tank. Issure which consequently rises builds up against the ram in lock- unit (734) of the swing grain tank unloading tube hydraulic (320) and in this process opens port A and/or B.		
	The return line of the hydraulic cylinder is relieved to the tank via the valve insert of the unactuated solenoid valve (Y34/Y33). The pressure rising further now opens the non-return valve (lock-up valve unit 734) at the opposite port B and/or A and the hydraulic cylinder is retracted or extended.			

Swinging the grain tank unloading tube Hydraulic cylinders



3-28

TIC	C	ominator 140 - 150	Hydraulic System
Key to diagram:	320	Swing grain tank unloading tu	ibe hydraulic cylinder
	S K	Securing wire Ram thread glued with liquid l	locking compound

Swinging the grain tank unloading tube Lock-up valve unit (734)



Key to diagram:

- А Hydraulic valve port В Hydraulic valve port A1 Consumer port B1
- Consumer port

Description of function:

Lock-up valve units (pilot controlled non-return valves) are used in order to lock functions while pressure is relieved and thus to ensure a fixed position of a consumer.

A rising pressure in port (B) moves the internal ram (K). This opens the opposite non-return valve in port A - the return line of the hydraulic cylinder to the tank is relieved (connection A-A1).

The continued pressure increase now opens the non-return valve in port B. The connection to consumer (B-B1) is relieved.

Reverse front attachment 3/2-way valve Y86 3.7



Dominator 140 - 150

Description of function:

Neutral

TIC

The spring force displaces the oil from the reversing cylinder (353) via the connection A-T in the solenoid valve (Y86) to the tank. During this process, port P1 is closed by the spool.

Reversing The solenoid valve (Y86) and the master valve (Y77) are actuated. The return line to the tank is now closed by the spool in solenoid valve (Y86) and the connection from P to the consumer port A is established. The reversing cylinder (353) now extends and swings the hydraulic motor (226) to the drive gearwheel. Just before reaching its end position, the oil flow from the reversing cylinder (353) to the hydraulic motor (226) is released, ensuring reliable gearwheel engaging. The non-return valve (732) keeps the hydraulic motor (226) from starting when pressure peaks occur in the return line.

Reverse front attachment Hydraulic cylinders



353

Key to diagram:

Reverse front attachment hydraulic cylinder

Description of function:

Reversing

Adjustment

When the solenoid valve (Y86) is actuated, the reversing cylinder (353) extends and swings the hydraulic motor (226) to the drive gearwheel. Just before reaching its end position, the oil flow from the reversing cylinder (353) to the hydraulic motor (226) is released. This ensures reliable engaging of the gearwheels for the reversing process.

The non-return valve (732) keeps the hydraulic motor (226) from starting when pressure peaks occur in the return line.

The reverser support is aligned towards the feed rake conveyor drive shaft by adjusting an eccentric bushing on the reverser cylinder (353). The piston stroke is adjusted using the set screw (E). With the reverser swung in, the set screw (E) must have a play of **0.5 mm** from the end stop, then jam the set screw (E).

Installation position of reverser motor OMP 200



Chapter 4

Ground drive hydraulics

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4.1

LINDE ground drive hydraulics circuit diagram

Key to diagram:

4.1 LINDE ground drive hydraulics circuit diagram



VI VII	Ground drive hydraulics hydrostatic pump Ground drive hydraulics hydrostatic motor
102 107 109 110	Pressure filter10 μm Oil drain Oil cooler Oil tank
205 207 209 211 218	Working hydraulics pump10.8 cmGround drive fixed displacement motor HMF 7575 cm³/rGround drive feed pump22.5 cmGround drive variable displacement pump HPV 7575 cm³/rSteering hydraulics pump6 cm³/r
301 313	3-D sieve pan hydraulic cylinder Ground drive servo control pump hydraulic cylinder
410	Orifice plateØ 1.5 m
601 606	3D sieve pan pendulum control 4/3 way valve Ground drive servo control 4/3 way valve
710 713 714 716 718 719 721	Ground drive filter bypass valve
902 903 904	Ground drive hydraulics high pressure backward measuring poly Ground drive hydraulics high pressure forward measuring poly Ground drive hydraulics feed pressure measuring point

Ground drive hydraulics hydrostatic pump

cs pump	10.8 cm ³ /rev.
d displacement motor HMF 75	75 cm ³ /rev.
1 pump	22.5 cm ³ /rev.
able displacement pump HPV 75	75 cm ³ /rev.
cs pump	6 cm ³ /rev.

nm

point oint

Description of function:

Oil supply	After starting the diesel engine, the ground drive feed pump (209) is driven. In this process, the oil quantity is taken from the housing. The housing is directly connected with the oil tank (110).
Feed pressure circuit	The feed pressure builds up from the oil quantity pumped through the oil cooler (109) and the pressure filter (102) by the ground drive feed pump (209) against the ground drive feed pressure relief valve (716). Depending on the spring setting, the oil flow is pre-stressed and then relieved to the tank.
	The feed pressure is applied at the combined ground drive multi-function valves (713/714) and at both servo cylinders via the ground drive servo adjustment valve (606). When the ground drive variable displacement pump (211) is not swung out, the feed pressure propagates to both sides of the high-pressure circuit via ground drive multi-function valves (713/714).
Servo control	 The cable mounted on the ground speed control lever moves the spool in the ground drive servo adjustment valve (606) from the neutral position to one or the other direction. Depending on the direction of travel, one of the ground drive pump servo adjustment hydraulic cylinders (313) is pressure-relieved whereas the other hydraulic cylinder remains connected to the feed pressure circuit. The motion at the swing disc corresponds to the pressure difference between the hydraulic cylinders. The ground drive pump servo adjustment hydraulic cylinders (313) swing the ground drive pump servo adjustment pump (211) only by the path defined by the ground speed control lever because there is a mechanical feedback of the swing angle to the ground drive servo adjustment valve (606). This mechanical feedback balances the spool in the ground drive servo adjustment valve (606) and therefore the pressure level between the two hydraulic cylinders at the control edge so that the defined swing angle is maintained.
High-pressure circuit	As soon as the ground drive variable displacement pump (211) is swung out, an axial motion is added to the radial motion of the pump unit. This axial motion displaces the oil in the cylinder space of the rotor and thus acts on the motor unit which converts this energy into a rotating motion by supporting against the fixed inclined disc.
	The respective suction side of the ground drive variable displacement pump (211) is pre-stressed via the feed pressure circuit and the corresponding ground drive multi-function valve (713/714). This ensures that the ground drive variable displacement pump (211) is sufficiently filled and that any occurring leaks are compensated.
	Since feed pressure is always applied on the suction side of the ground drive variable displacement pump (211) as well as on the return flow side of the ground drive fixed displacement motor (210), this area is referred to as low-pressure side within the high-pressure circuit.

Description of function:				
High-pressure limitation	If the system pressure rises above the set maximum value, this overpressure is relieved to the feed pressure circuit by the ground drive multi-function valves (713/714). The high-pressure limitation should only respond for a short time during operation since the large oil flow which has to be displaced by the heavily pre-stressed valves would rapidly overheat the system.			
Flushing device	The respective high-pressure side in the high-pressure circuit actuates the ground drive purging shuttle valve (721) in the ground drive fixed displacement motor (210) so the corresponding low-pressure side has a connection to the motor housing via the ground drive purge pressure control valve (719). Since the pressure setting of the ground drive purge pressure control valve (719) is lower than that of the ground drive feed pressure relief valve (716), a constant oil quantity is exchanged by the ground drive feed pump (209) via the restrictor in the ground drive purge pressure control valve (719).			

4.2 Pump unit


Key to diagram:

- B Control bottom D Shaft seal
- D Shaft se G Slide
- K Ram
- L Bearing
- R Cylinder rotor
- S Swing disc
- V Adjusting lever
- W Drive shaft

Description of function:

As soon as the diesel engine is started, the cylinder rotor (R) as well as the ground drive feed pump (209) are driven by the nine pistons (K) arranged radially around the drive shaft (W).

In this process, the pistons (K) are pressed against the swing disc (S) by means of the slides (G) due to the feed pressure applied on both sides of the high-pressure circuit (H).

One of the servo cylinders is actuated by the ground drive servo adjustment valve (606) so that this cylinder swings the swing disc (S) according to the direction of travel and the ground speed.

During the swinging motion, the pistons (K) make an axial movement on the inclined plane of the swing disc (S) which results in the oil in the filled cylinder space being displaced and in a pressure building up against the resistance at the motor.

When the entire oil quantity in the cylinder space has been displaced, the piston (K) rotating with the rotor (R) is pushed back by the feed pressure and against the sloping inclined plane of the swing disc (S) on the low-pressure side.

The cylinder spaces in the rotor (R) are thus filled one after the other on the sloping side of the swing disc (S) (low pressure) and then displace this oil quantity on the rising side (high pressure) against the motor unit.

According to the direction of travel, the swing disc (S) is moved to one or the other direction, making high pressure and low pressure change sides as well. The ground speed depends on the oil flow quantity and consequently on the swing angle of the swing disc (S). The swing angle pre-set on the ground speed control lever is maintained by the mechanical feedback from the swing disc (S) to the servo control valve (606).

The low-pressure side is separated from the high-pressure side inside the pump unit above the control bottom (B). For sealing purposes, the cylinder rotor (R) is pushed against the control bottom (B) only by a compression spring.

The exact return of the swing disc to its neutral position is achieved by compressed springs, and this factory setting cannot be modified from the outside.

The position of the adjusting lever (V) on the shaft gearing is marked with a punch blow on the servo adjustment housing. This position corresponds to the neutral position of the servo adjustment valve which is achieved within an angle of 8° of the adjusting range.

4.3 Servo control valve



Key to diagram:	313 606	Ground drive servo control pump hydraulic cylinder Ground drive servo adjustment valve			
	G M P S V	Threaded bushing Mechanical feedback Spool Swing disc Adjusting lever			
Description of function:					
Servo control valve	In the neutral position of the ground drive servo adjustment valve (606), both ground drive pump servo control hydraulic cylinders (313) are pressure-loaded, keeping the swing disc (S) stable in any position.				
	The cab ground of ground of pressure the feed The mon difference cylinder	The cable mounted on the adjusting lever (V) moves the spool (P) in the ground drive servo adjustment valve (606) from the neutral position to one or the other direction. Depending on the direction of travel, one of the ground drive pump servo adjustment hydraulic cylinders (313) is thus pressure-relieved whereas the other servo cylinder remains connected to the feed pressure circuit. The movement at the swing disc (S) thus corresponds to the pressure difference between the ground drive pump servo adjustment hydraulic cylinders (313).			
	The gro the varia lever (V to the gr This me servo ac The pre in both g	und drive pump servo adjustment hydraulic cylinders (313) swing able displacement pump only by the path defined by the adjusting) because there is a mechanical feedback (M) of the swing angle round drive servo adjustment valve (606). Echanical feedback (M) balances the spool (P) in the ground drive djustment valve (606) at the control edge to the neutral position. -set swing angle is thus maintained by the pressure compensation ground drive pump servo adjustment hydraulic cylinders (313).			
Adjusting the hydraulic neutral position:	To align hydrauli in the se To do th from the pressure determin (G) to ou the bush position	the mechanical neutral position of the adjusting lever (V) with the c neutral position of the variable displacement pump, the spool (P) ervo adjustment valve is adjusted using the threaded bushing (G). his, the bushing (G) is first set to a clearance of $X = 14.75 \text{ mm}$ (X) a housing of the ground drive servo adjustment valve (606). A e measurement on both sides of the high-pressure circuit nes the respective pressure rise caused by rotating the bushing ne or the other direction. The centre position of the path by which hing (G) has been rotated corresponds to the average neutral .			

4.4 Ground drive multi-function valve



Key to diagram:	1 2 3 4	Valve plunger High-pressure spring Valve insert Feed spring	
	H N S	High pressure No high pressure Feed pressure	
Description of function:			
High-pressure limitation	High p valve o When spring pressu	High pressure (H) is applied to the valve plunger (1) via the bores in the valve cartridge. When the system pressure exceeds to pre-set value of the high-pressure spring (2), the valve plunger (1) backs away to the bottom against the spring pressure and relieves the high-pressure side towards the feed pressure circuit (S).	
Feed	As soc cartrid upwar circuit	on as there is no high pressure (N) applied against the valve ge, the feed pressure (S) presses the entire valve insert (3) ds against the feed spring (4) and thus opens the feed pressure (S) to the low-pressure side (N).	

4.5 Ground drive fixed displacement motor



Key to diagram:	210	Ground drive fixed displacement motor HMF 75
	721 719	Ground drive flush-out shuttle valve Ground drive flush pressure control valve
	B D G K L R S W	Control bottom Shaft seal Slide Ram Bearing Cylinder rotor Fixed inclined disc Driven shaft

Description of function:

hydraulics circuit diagram

See also ground drive

As soon as the diesel engine is started, the feed pump in the pump unit is also driven. In this process, the pistons (K) in the cylinder rotor (R) of the motor unit are pressed against the fixed inclined disc (S) by means of the slides (G) due to the feed pressure applied on both sides of the high-pressure circuit.

As soon as the variable displacement pump is swung out, the pressure builds up against the nine pistons (K) in the cylinder rotor (R) which is geared to the driven shaft (W), one after the other. Here the pistons (K) support themselves against the inclined plane of the fixed inclined disc (S) and thus convert this energy into a rotating motion against the resistance at the driven shaft (W).

The direction of rotation here depends on the direction of the oil flow and thus on the swing direction of the variable displacement pump, with high pressure and low pressure changing the sides. The motor speed results from the oil flow quantity therefore from the swing angle of the variable displacement pump.

The low-pressure side is separated from the high-pressure side inside the motor unit above the control bottom (B). For sealing purposes, the cylinder rotor (R) is pushed against the control bottom (B) only by a compression spring.

The respective high-pressure side in the high-pressure circuit actuates the ground drive purging shuttle valve (721) in the fixed displacement motor so the corresponding low-pressure side has a connection to the motor housing via the ground drive purge pressure control valve (719). Since the pressure setting of the ground drive purge pressure control valve (719) is lower than that of the feed pressure relief valve, a constant oil quantity is exchanged by the feed pump (209) via the restrictor in the ground drive purge pressure control valve (719).

4.6 Maintenance

Filling instructions	Engage 3 rd gear at the gearshift lever Disengage all-wheel drive Apply parking brake Connect pressure gauges on both high-pressure sides (M1+M2) Fill tank with hydraulic oil Pull out engine cut-off system relay and/or cable jumper Crank the diesel engine for a short period using the electric starting motor Check and correct the oil level Repeat procedure until the pressure has risen by approx. 10 bar Re-install engine cut-off system relay and/or cable jumper Start diesel engine at min. no-load speed Load the system with 50-150 bar forward for approx. 1 minute Load the system with 50-150 bar backward for approx. 1 minute Shut off diesel engine Check and correct the oil level if necessary Set gearshift lever to neutral position Start diesel engine at min. no-load speed Swing variable displacement pump forward for approx. 2 minutes Shut off diesel engine Check and correct the oil level if necessary Set gearshift lever to neutral position
Inspection regulations	Connect pressure gauges on both high-pressure sides Apply parking brake Heat up the system to an operating temperature of approx. 60°C Move ground speed control lever to neutral position Set diesel engine to max. no-load speed Measure the feed pressure: 19 bar Feed pressure difference on both sides: max. 3 bar Swing out the variable displacement pump fully to one direction Pressure drop on low-pressure side: max. 4 bar Set diesel engine to min. no-load speed Engage 3 rd gear in manual transmission Apply service brake Slowly swing the variable displacement pump forward for 5 sec. max. High pressure measurement: 420 to 450 bar Low pressure measurement: min. 14 bar Slowly swing the variable displacement pump backward for 5 sec. max. High pressure measurement: 420 to 450 bar Low pressure measurement: 420 to 450 bar Slowly swing the variable displacement pump backward for 5 sec. max.

4.7 3D cleaning system



Key to diagram:

1 Control head 2 Pendulum 3 Cup Spool

4

Description of function:

Volumetric flow is applied to port (P) of the control head. When the pendulum (2) is in the centre position, P is blocked at the spool (4). When the pendulum changes its position to the right or left, relative to the cup, the spool (4) is moved. In this process, connections are made between P and A as well as B and R or between P and B as well as A and R, depending on the direction in which the pendulum moves.

Position of components

TIC

5-k-20











5-i-18











TIC

7-j-19











3-o-19



3-0-19



3-0-19



R-4

7-j-19



3-o-19







6-n-20



4-k-16



4-m-20



7-r-18









441

6-n-20



5-i-18

4-k-16



5-i-18



Component grid



Component grid

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Hydraulic System

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